# Kinematics $C^{\circ}$ Dynamics of Linkages Lecture G: Planetary Gears 

## Planetary Gear Trains

## Planetary gearset: A planet gear orbits around a sun gear



## Planetary Gear Trains

- A planetary set is a 2OLF system requiring 2 inputs (arm and sun)
- If either is held, the set is IDEF which is the conventional case


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## Useful planetary configuration

- A more useful configuration is where a ring gear is added
- This will bring the gear output back to a grounded pivat $\left(\mathrm{D}_{2}\right)$



## Planetary transmission



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## Levai's planetary gearsets configurations



## Planetary gearsets equations

$$
\begin{aligned}
& \omega_{\text {gear }}=\omega_{\text {arm }}+\omega_{\text {gear/arm }} \\
& m_{v}=\frac{\omega_{\text {out }}}{\omega_{\text {in }}}= \pm \frac{d_{\text {in }}}{d_{\text {out }}}= \pm \frac{N_{\text {in }}}{N_{\text {out }}} \\
& \frac{\omega_{\text {gear } 1}}{\omega_{\text {gear } 2}}= \pm \frac{N_{\text {gear } 2}}{N_{\text {gear } 1}} \\
& \frac{\omega_{\text {gear } 1 / a r m}}{\omega_{\text {gear } 2 / a r m}}= \pm \frac{N_{\text {gear } 2}}{N_{\text {gear } 1}}
\end{aligned}
$$



## Example 1

Find the absolute output angular velocity of the ring gear
Sun gear

$$
\begin{aligned}
& \mathrm{N}_{2}=40 \\
& \mathrm{~N}_{3}=20 \\
& \mathrm{~N}_{4}=80
\end{aligned}
$$

Planet gear
Ring gear

Input to arm
Input to sun
200 rpm cw
100 rpm cw


## Example 1 Solution - Tabular method

Start by finding the missing values in the table below

| Gear \# | $\boldsymbol{\omega}_{\text {gear }}=$ | $\boldsymbol{\omega}_{\text {arm }} \boldsymbol{+}$ | $\boldsymbol{\omega}_{\text {gear/arm }}$ |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Gear ratio |  |  |  |  |  |  |
| 2 | -100 | -200 | a | $-40 / 20$ |  |  |  |
| 3 | d | -200 | b | $+20 / 80$ |  |  |  |
| 4 | e | -200 | c |  |  |  |  |

## Example 1 Solution - Tabular method

Solving for a, b, c, d, and e we get:
$a=-100+200=100$
$\frac{b}{a}=-\frac{40}{20} \rightarrow b=-a \frac{40}{20}=-200$
$\frac{c}{b}=+\frac{20}{80} \rightarrow c=+b \frac{20}{80}=-50$
$d=-200-200=-400$
$e=-200-50=-250$

| Gear \# | $\boldsymbol{\omega}_{\text {gear }}=$ | $\boldsymbol{\omega}_{\text {arm }}+$ | $\boldsymbol{\omega}_{\text {gear/arm }}$ |  |
| :---: | :---: | :---: | :---: | :---: |
| 2 | -100 | -200 | a | Gear ratio |
| 3 | d | -200 | b | $-40 / 20$ |
| 4 | e | -200 | c | $+20 / 80$ |


| Gear \# | $\omega_{\text {gear }}=$ | $\omega_{\text {arm }}{ }^{+}$ | $\boldsymbol{\omega}_{\text {gear/arm }}$ |  |
| :---: | :---: | :---: | :---: | :---: |
| 2 | -100 | -200 | +100 | Gear ratio |
| 2 | -100 | -200 | $+100$ | -40/20 |
| 3 | -400 | -200 | -200 | +20/80 |
| 4 | -250 | -200 | -50 |  |

The absolute output angular velocity of the ring gear is $\mathbf{2 5 0} \mathbf{~ w}$

## Example I Solution - Formula method

Similar to compound gears but with relative velocities.
$\omega_{F}$ the angular velocity of the first gear
$\omega_{L}$ the angular velocity of the last gear
For the first gear $\omega_{F / a r m}=\omega_{F}-\omega_{\text {arm }}$
For the last gear $\omega_{L \text { Larm }}=\omega_{L}-\omega_{\text {arm }}$
$\frac{\omega_{\text {out }}}{\omega_{\text {in }}}=\frac{\omega_{\text {L/arm }}}{\omega_{F / \text { arm }}}=\frac{\omega_{L}-\omega_{\text {arm }}}{\omega_{F}-\omega_{\text {arm }}}= \pm \frac{\text { product of Number of teeth of driver gears }}{\text { product of Number of teeth of driven gears }}$
Both first and last gear must be pivoted to ground, and there must be a path of meshes connecting them.

## Example 1 Solution - Formula methad

$\omega_{F}=\omega_{2}=-100$
$\omega_{L}=\omega_{4}$ to be found
For the first gear $\omega_{\text {F/arm }}=\omega_{F}-\omega_{\text {arm }}=-100-(-200)=100$
For the last gear $\omega_{\text {L/arm }}=\omega_{L}-\omega_{\text {arm }}=\omega_{4}-(-200)=\omega_{4}+200$
$\frac{\omega_{L / a r m}}{\omega_{F / a r m}}=\frac{\omega_{L}-\omega_{a r m}}{\omega_{F}-\omega_{a r m}}=\left(-\frac{N_{2}}{N_{3}}\right)\left(+\frac{N_{3}}{N_{4}}\right)=\left(-\frac{40}{20}\right)\left(+\frac{20}{80}\right)=-\frac{1}{2}$
$\frac{\omega_{L}-\omega_{\text {arm }}}{\omega_{F}-\omega_{\text {arm }}}=\frac{\omega_{4}+200}{100}=-\frac{1}{2} \quad \rightarrow \quad \omega_{4}=-250$
The absolute output angular velocity of the ring gear is $\mathbf{2 5 0} \mathbf{~ w}$

## Example 2



$$
\begin{array}{ll}
N_{2}=30 & N_{3}=25 \quad N_{4}=45 \quad N_{5}=30 \quad N_{6}=160 \\
\omega_{6}=40 & \omega_{\text {arm }}=-50 \quad \text { Find } \omega_{2} \text { and } \omega_{3}
\end{array}
$$

## Example 2 Solution

$\omega_{F}=\omega_{2}$ to be found
$\omega_{L}=\omega_{6}=40$
For the first gear $\omega_{F / \text { arm }}=\omega_{F}-\omega_{\text {arm }}=\omega_{2}-(-50)=\omega_{2}+50$
For the last gear $\omega_{L / a r m}=\omega_{L}-\omega_{\text {arm }}=40-(-50)=90$
$\frac{\omega_{\text {L/arm }}}{\omega_{\text {F/arm }}}=\frac{\omega_{L}-\omega_{\text {arm }}}{\omega_{F}-\omega_{\text {arm }}}=\left(-\frac{N_{2}}{N_{4}}\right)\left(-\frac{N_{3}}{N_{5}}\right)\left(\frac{N_{5}}{N_{6}}\right)=\left(-\frac{30}{45}\right)\left(-\frac{25}{30}\right)\left(\frac{30}{160}\right)=\frac{5}{48}$
$\frac{\omega_{L}-\omega_{\text {arm }}}{\omega_{F}-\omega_{\text {arm }}}=\frac{90}{\omega_{2}+50}=\frac{5}{48} \quad \rightarrow \quad \omega_{2}=814$
The absolute output angular velocity of the first gear is $\mathbf{8 1 4}$ cew

## Example 2 Solutian

$\omega_{3}=\omega_{4}$ on the sameshaft
$\frac{\omega_{4}-\omega_{\text {arm }}}{\omega_{2}-\omega_{\text {arm }}}=\frac{\omega_{3}-\omega_{\text {arm }}}{\omega_{2}-\omega_{\text {arm }}}=-\frac{N_{2}}{N_{4}}=-\frac{30}{45}$

$\omega_{3}=-\frac{N_{2}}{N_{4}}\left(\omega_{2}-\omega_{\text {arm }}\right)+\omega_{\text {arm }}=-\frac{30}{45}(814+50)-50 \rightarrow \omega_{3}=-626$
The absolute output angular velocity of the first gear is 626 cw

## SolidWorks Simulation



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